

Research Practice, Vibrations, Nonlinearities, Life :)
Seminar to Structures Group, IIST Thiruvananthapuram

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Indian Institute of Technology Madras, Chennai 600036, IN

February 6, 2025

Table of Contents

- 1 Academia as a Career: Reflections
 - My Journey So Far
 - Reflections on Life in Academia
 - Opportunities
- 2 Highlights from my Research
- 3 Nonlinear Dynamics of Jointed Structures
 - Physics-Based Modeling of Bolted Joints
 - Self-Excited Oscillations
 - Research Plan Overview
 - Focus on Free and Open Source Software

1.1. My Journey So Far

Academia as a Career: Reflections



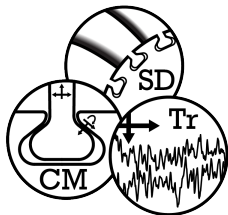
RICE UNIVERSITY



University of Stuttgart
Germany

Brief Bio

- BTech Aerospace (2017), **IIST**
- MS (2019) & PhD (2021) Mechanical, **Rice University**, Houston (adv: Dr. Matthew Brake) + Postdoc (2021-22)
- Humboldt Postdoctoral Researcher at **University of Stuttgart**, Stuttgart, DE (2022-24)



1.2. Reflections on Life in Academia

Academia as a Career: Reflections

- Life as a PhD student, as a Postdoc, as a faculty member

1.2. Reflections on Life in Academia

Academia as a Career: Reflections

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- **Time is Fragmented**

1.2. Reflections on Life in Academia

Academia as a Career: Reflections

- Life as a PhD student, as a Postdoc, as a faculty member
- **Time is Fragmented**
- Academia in general

1.2. Reflections on Life in Academia

Academia as a Career: Reflections

- Life as a PhD student, as a Postdoc, as a faculty member
- **Time is Fragmented**
- Academia in general
- Academia in India, US, Europe

1.3. Opportunities

Academia as a Career: Reflections

- Opportunities are plenty!

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Academia as a Career: Reflections

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- There are excellent postdoctoral fellowships (Humboldt, Marie-Curie, JSPS, George Foster, Fullbright, etc.). **Please network and find out!**

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- Key is to work on skills right now. **PhDs in engineering is not (yet) saturated.**

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Academia as a Career: Reflections

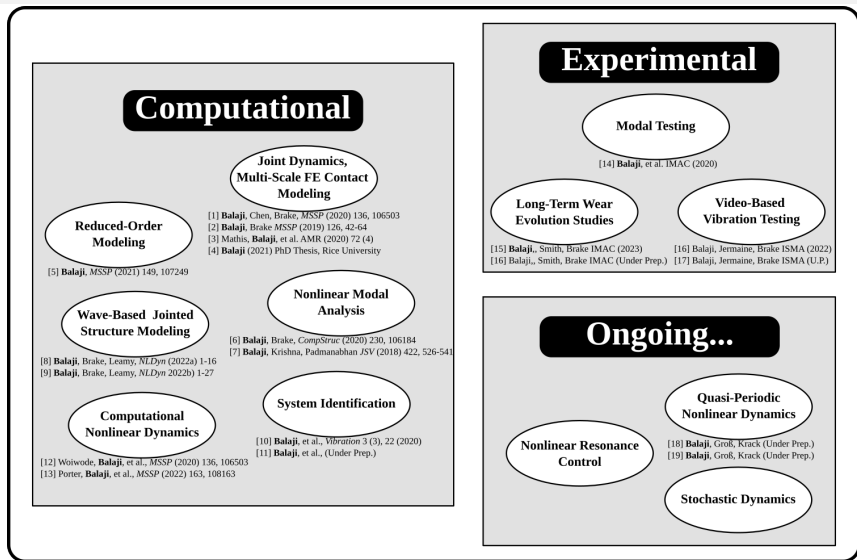
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1.3. Opportunities

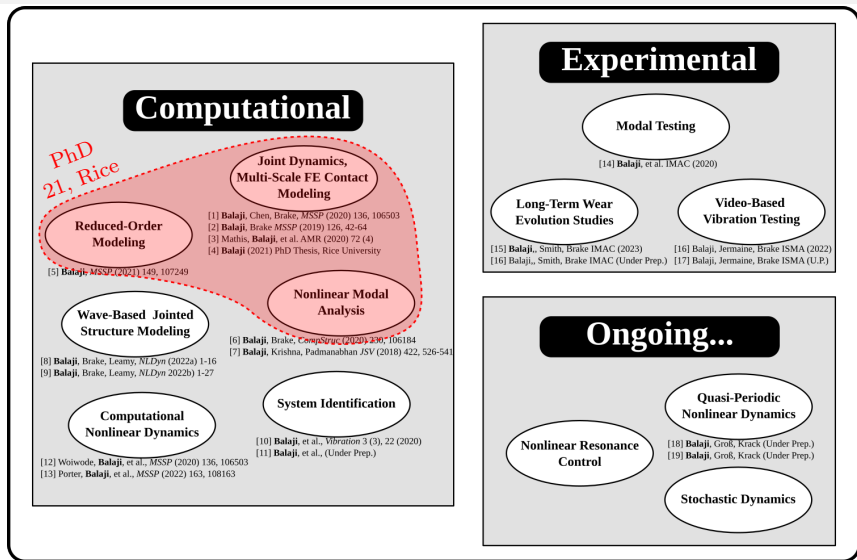
Academia as a Career: Reflections

- Opportunities are plenty!
- There are excellent postdoctoral fellowships (Humboldt, Marie-Curie, JSPS, George Foster, Fullbright, etc.). **Please network and find out!**
- Key is to work on skills right now. **PhDs in engineering is not (yet) saturated.**
- **Most IIT's are mandated to grow.** New IIT's coming up. **Engineering faculty positions are not beyond your reach!**
- **Research-wise**, Indian academia is quite comfortable.

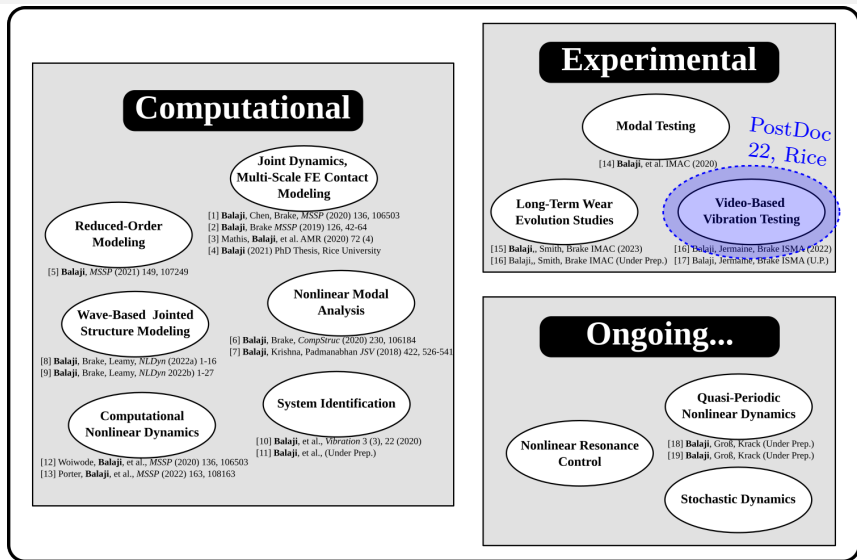
2. Highlights from my Research



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2. Highlights from my Research

Comp

Reduced-Order Modeling

[5] Balaji, MSSP (2021) 149, 107249

Wave-Based Jointed Structure Modeling

[8] Balaji, Brake, Leamy, *NLDyn* (2022a) 1-16
 [9] Balaji, Brake, Leamy, *NLDyn* 2022b) 1-27

Computational Nonlinear Dynamics

[12] Woiwode, Balaji, et al., MSSP (2020) 136, 1
 [13] Porter, Balaji, et al., MSSP (2022) 163, 106163

Digital Image Correlation

$\phi : \Omega \rightarrow \Omega'$

$$\begin{Bmatrix} X \\ Y \end{Bmatrix} + \begin{Bmatrix} \Delta x \\ \Delta y \end{Bmatrix} \xrightarrow{\phi} \begin{Bmatrix} X \\ Y \end{Bmatrix} + \begin{Bmatrix} u \\ v \end{Bmatrix} + \begin{bmatrix} u_x & u_y \\ v_x & v_y \end{bmatrix} \begin{Bmatrix} \Delta x \\ \Delta y \end{Bmatrix} + \frac{1}{2} \begin{bmatrix} u_{xx} & u_{xy} & u_{yy} \\ v_{xx} & v_{xy} & v_{yy} \end{bmatrix} \begin{Bmatrix} \Delta x^2 \\ 2\Delta x \Delta y \\ \Delta y^2 \end{Bmatrix}$$

Experimental

Video-Based Vibration Testing

[16] Balaji, Jermaine, Brake ISMA (2022) Prep.) [17] Balaji, Jermaine, Brake ISMA (U.P.)

Quasi-Periodic Nonlinear Dynamics

[18] Balaji, Groß, Krack (Under Prep.)
 [19] Balaji, Groß, Krack (Under Prep.)

Stochastic Dynamics

al Testing
 et al. IMAC (2020)
 PostDoc
 22, Rice

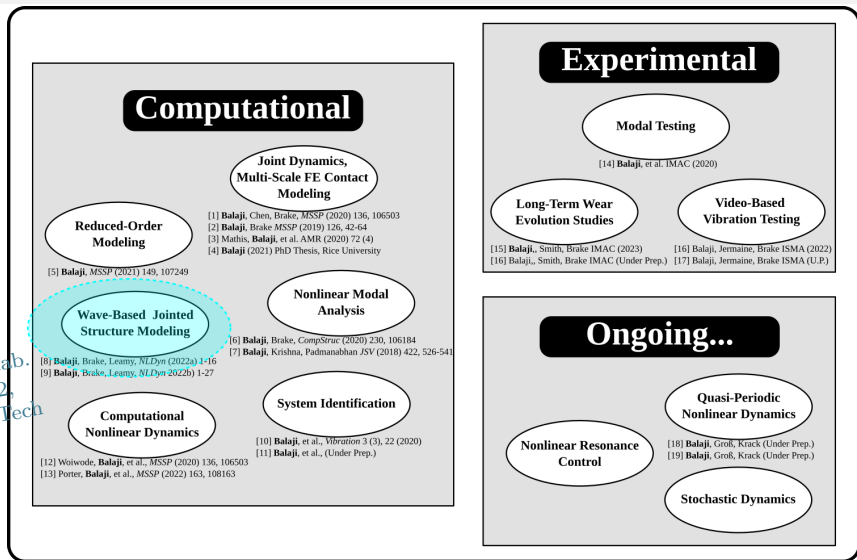
Video-Based Vibration Testing

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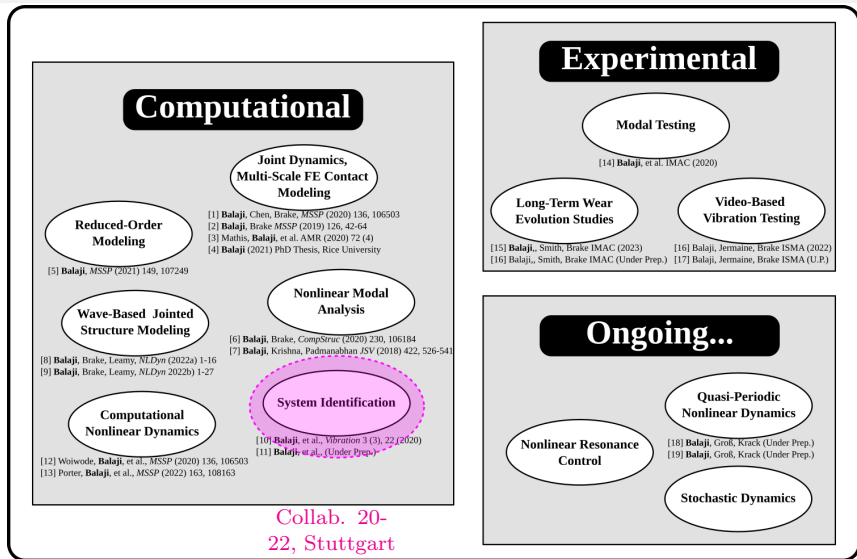
Quasi-Periodic Nonlinear Dynamics

Stochastic Dynamics

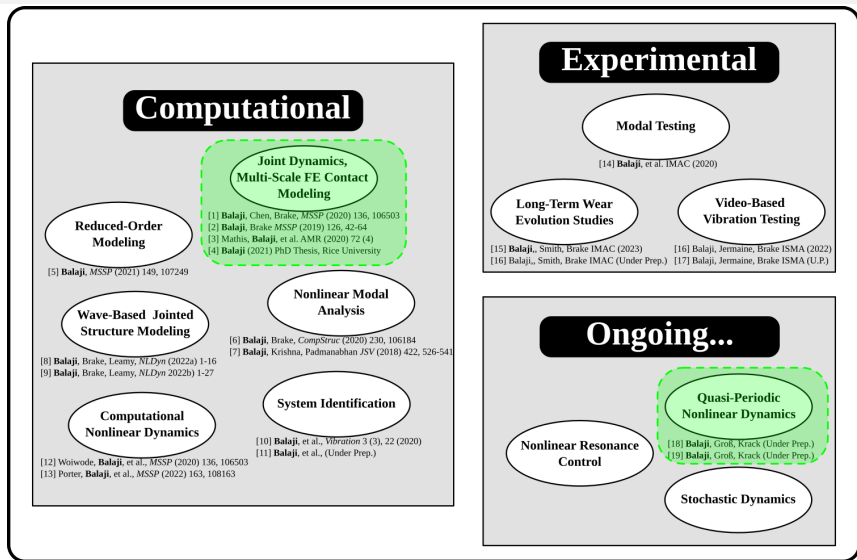
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2. Highlights from my Research



3. Nonlinear Dynamics of Jointed Structures

3.1. Physics-Based Modeling of Bolted Joints



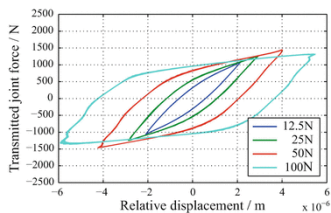
The Gaul Resonator (Süß, Janeba, and Willner 2018)

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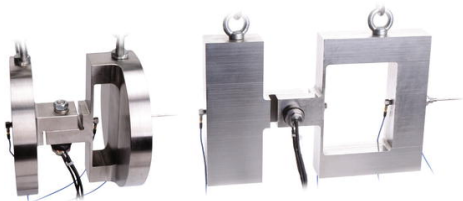


The Gaul Resonator (Süß, Janeba, and Willner 2018)

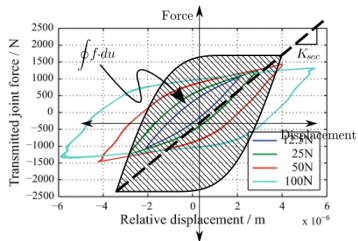


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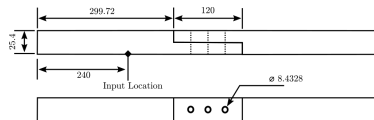


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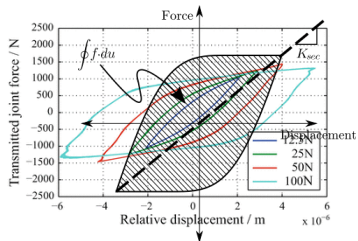
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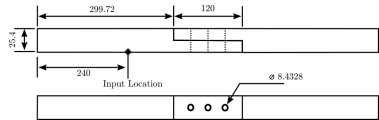
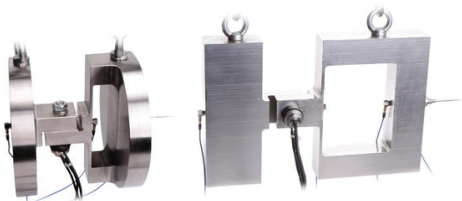
The Brake-Reuß Beam (Brake and Reuß 2018)

The Gaul Resonator (Süß, Janeba, and Willner 2018)



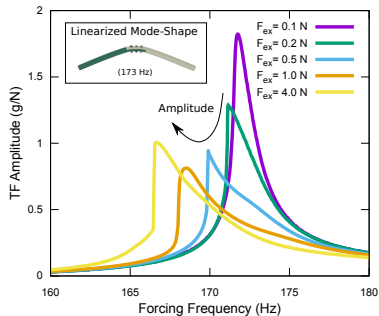
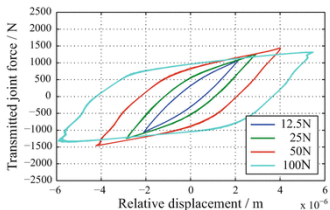
3. Nonlinear Dynamics of Jointed Structures

3.1. Physics-Based Modeling of Bolted Joints



The Brake-Reuß Beam (Brake and Reuß 2018)

The Gaul Resonator (Süß, Janeba, and Willner 2018)



3. Nonlinear Dynamics of Jointed Structures

3.1. Physics-Based Modeling of Bolted Joints: Nonlinear Modal Analysis

Rayleigh Quotient-based NMA

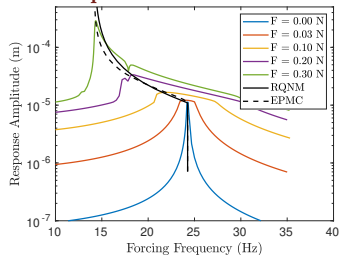
- The NMA is posed as an eigenvector-dependent non-linear Eigenvalue Problem (NEPv)

$$\underline{K} \underline{u} + \underline{f}_{nl}(\underline{u}, \dots) - \underline{f}_s - \lambda \mathbf{M}(\underline{u} - \underline{u}_s) = 0$$

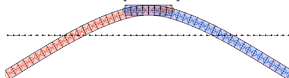
$$(\underline{u} - \underline{u}_s)^T \mathbf{M}(\underline{u} - \underline{u}_s) - q^2 = 0.$$

- See (Balaji and Brake 2020) for details

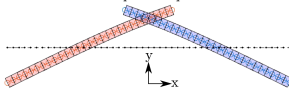
Example: Frictional Beam



Modal Amplitude: $q = 10^{-6.00}$

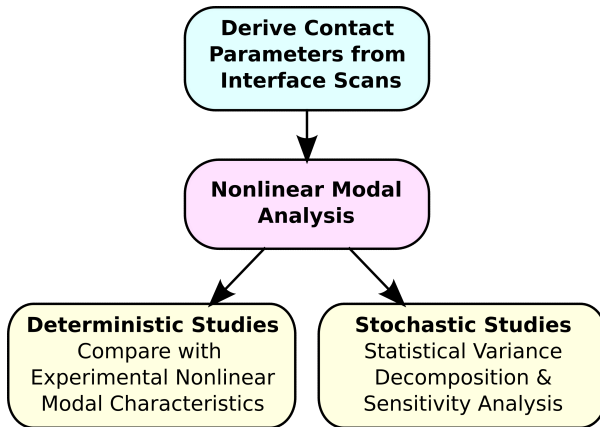


Modal Amplitude: $q = 10^{-2.00}$



3. Nonlinear Dynamics of Jointed Structures

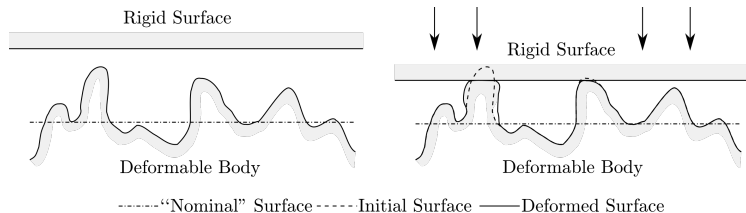
3.1. Physics-Based Modeling of Bolted Joints



The TriboMechadynamics Approach

3. Nonlinear Dynamics of Jointed Structures

3.1. Physics-Based Modeling of Bolted Joints



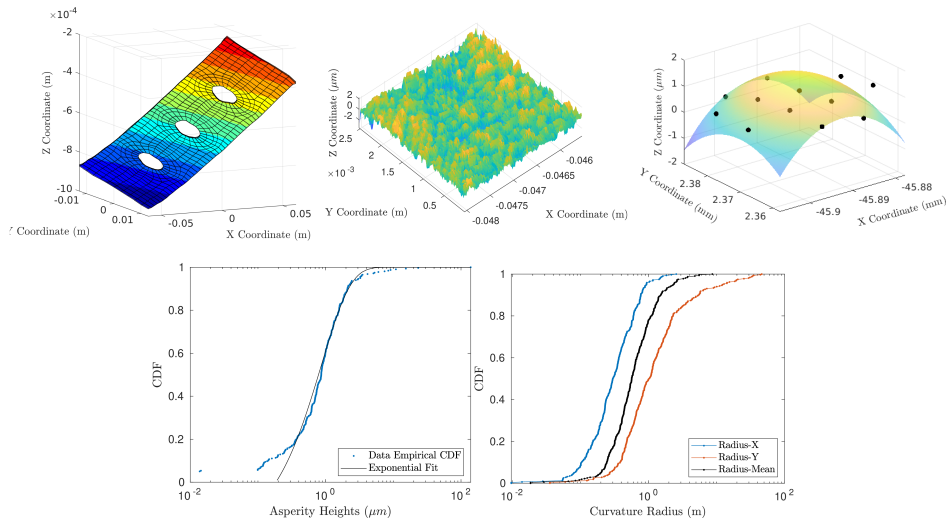
- The statistical treatment of rough contact has been popular in the contact mechanics community from (Greenwood and Williamson 1966)
- The idea is to describe the reaction force as a statistical expectation of asperity-reaction forces randomly distributed over a given surface

Exponentially Distributed Surface

- The asperity heights are fitted to a two parameter exponential distribution, following (Polycarpou and Etsion 1999; Medina, Nowell, and Dini 2013)

3. Nonlinear Dynamics of Jointed Structures

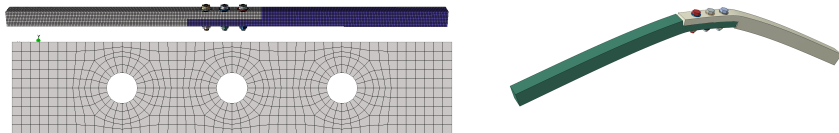
3.1. Physics-Based Modeling of Bolted Joints: Contact Parameter Estimation



3. Nonlinear Dynamics of Jointed Structures

3.1. Physics-Based Modeling of Bolted Joints: Modeling Methodology

Linear finite element model with only contact non-linearities



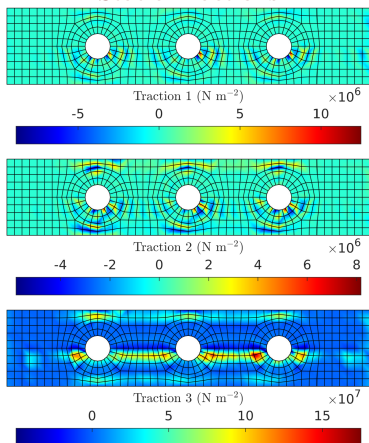
Factors for Uncertainty Propagation

S.No.	Description	Symbol	Distribution	Quadrature
1.	Coefficient of Friction	μ	Exponential (mean ≈ 0.1183)	Gauss-Laguerre
2.	Gap Function	g	Normal (fit parameters)	Gauss-Hermite
3.	Asperity height exp.	λ	Normal (fit parameters)	Gauss-Hermite
4.	Mean Radius	R	Normal (fit parameters)	Gauss-Hermite
5.	Stage Rotation X	θ_X	Normal (0 mean, 15° s.d.)	Gauss-Hermite
6.	Stage Rotation Y	θ_Y	Normal (0 mean, 15° s.d.)	Gauss-Hermite
7.	Bolt Prestress Force	P	Normal (exp. mean, s.d.)	Gauss-Hermite

3. Nonlinear Dynamics of Jointed Structures

3.1. Physics-Based Modeling of Bolted Joints: Mean Model Results

Static Traction



Prediction of Linearized Natural Frequency

S.No.	Exp. (Hz)	Mean Model (Hz)	Error (%)
1	179.56	179.41	0.0845
2	594.71	594.72	0.0016
3	1199.8	1197.1	0.2209

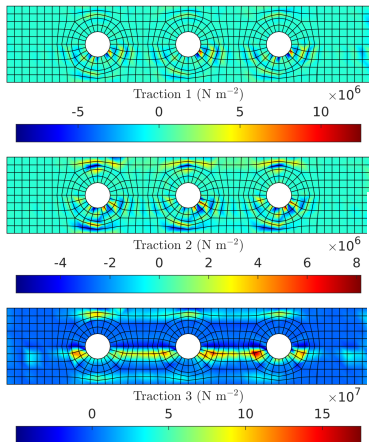
The interfaces after several hours of testing



3. Nonlinear Dynamics of Jointed Structures

3.1. Physics-Based Modeling of Bolted Joints: Mean Model Results

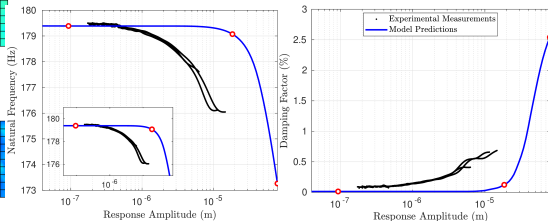
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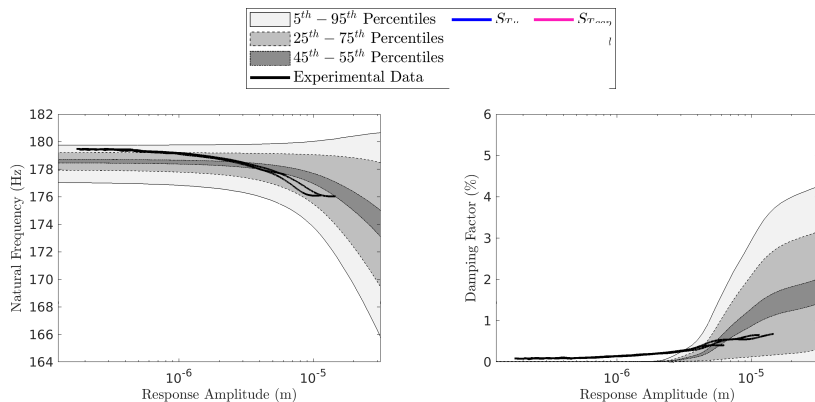
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Mode 1 Dynamics

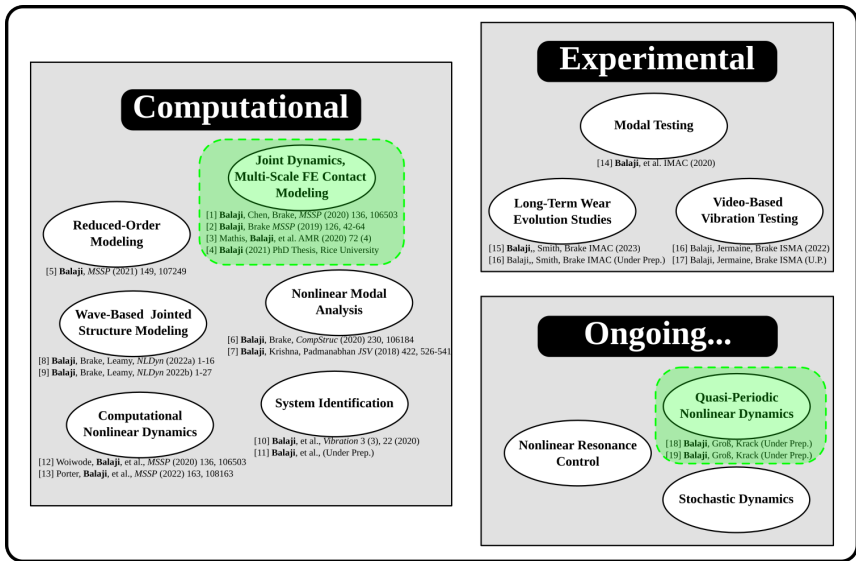


3. Nonlinear Dynamics of Jointed Structures

3.1. Physics-Based Modeling of Bolted Joints: PCE Results



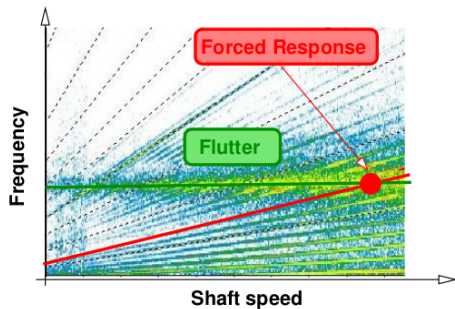
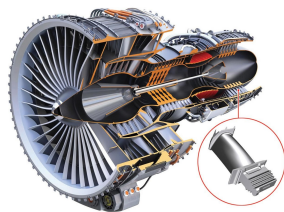
3. Nonlinear Dynamics of Jointed Structures



3. Nonlinear Dynamics of Jointed Structures

3.2. Self-Excited Oscillations

- Multiple sources of excitation in jet-engines



Blade stage vibration data from MTU aero test engines (Corral, Gallardo, and Ivaturi 2013)

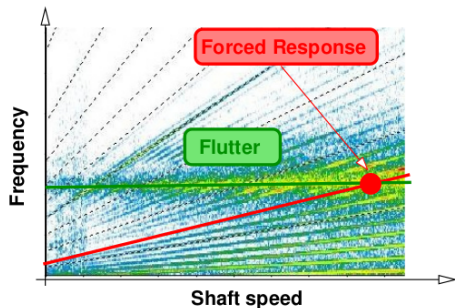
3. Nonlinear Dynamics of Jointed Structures

3.2. Self-Excited Oscillations

- Multiple sources of excitation in jet-engines
- The aerodynamic interactions can sometimes be modeled as a **self-excitation** for a traveling wave mode

$$\ddot{\eta} + c\dot{\eta} + \omega_0^2\eta = 2\omega_0\zeta\dot{\eta}$$

$$\implies \ddot{\eta} + (c-2\omega_0\zeta)\dot{\eta} + \omega_0^2\eta = 0$$



Blade stage vibration data from MTU aero test engines (Corral, Gallardo, and Ivaturi 2013)

3. Nonlinear Dynamics of Jointed Structures

3.2. Self-Excited Oscillations

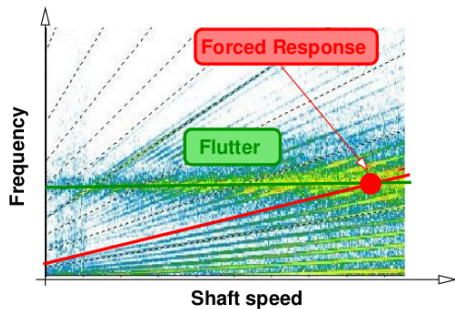
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- These structures are often supported by frictional contacts and the “modal” equations become

$$\ddot{\eta} + (c-2\omega_0\zeta)\dot{\eta} + \omega_0^2\eta + f_{nl}(\eta, \dots) = 0.$$



Blade stage vibration data from MTU aero test engines (Corral, Gallardo, and Ivaturi 2013)

3. Nonlinear Dynamics of Jointed Structures

3.2. Self-Excited Oscillations: A Self-Excited SDOF Benchmark

- A representative problem is analyzed

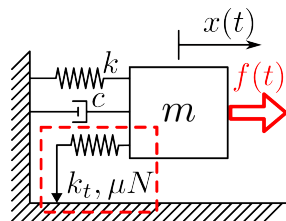
$$m\ddot{x} - c\dot{x} + kx + f_{nl}(x, \dots) = \frac{F}{2}e^{j\Omega t} + c.c.$$

using a Harmonic Balance approach, with the Fourier ansatz:

$$x(t) = \sum_{k \in \mathcal{H}} U_k e^{jk\Omega t} + c.c.$$

- Quasi-Periodic HB Ansatz ($\tau_i = \Omega_i t$):

$$x(t) = \sum_{\underline{k} \in \mathcal{H}} U_{\underline{k}} \exp(i(k_1 \tau_1 + k_2 \tau_2))$$



3. Nonlinear Dynamics of Jointed Structures

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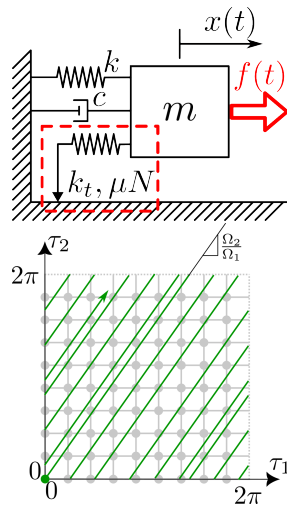
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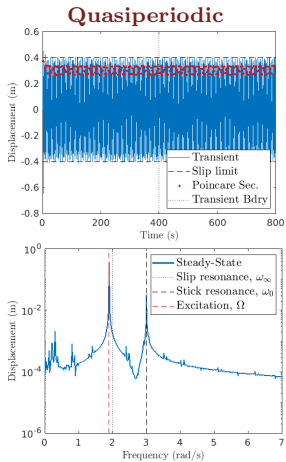
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3. Nonlinear Dynamics of Jointed Structures

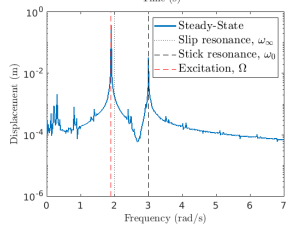
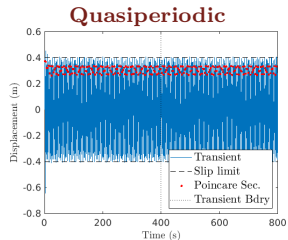
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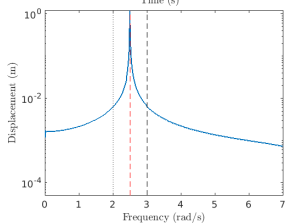
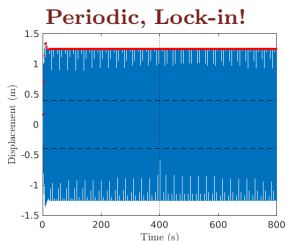
$$\Omega < \omega_{res}$$

3. Nonlinear Dynamics of Jointed Structures

3.2. Self-Excited Oscillations: A Self-Excited SDOF Benchmark



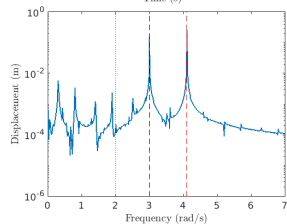
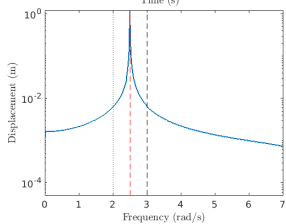
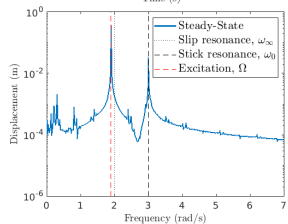
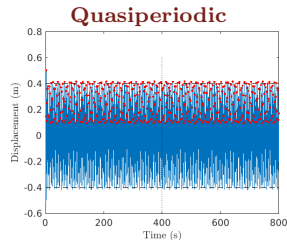
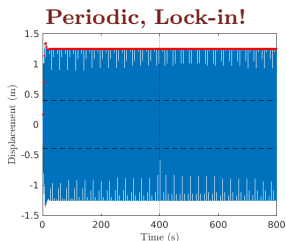
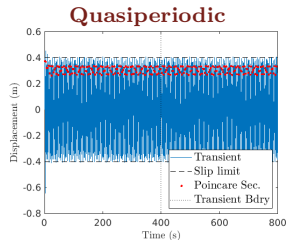
$$\Omega < \omega_{res}$$



$$\Omega \sim \omega_{res}$$

3. Nonlinear Dynamics of Jointed Structures

3.2. Self-Excited Oscillations: A Self-Excited SDOF Benchmark



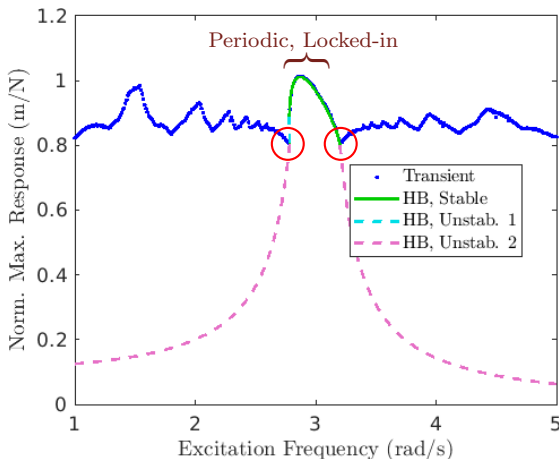
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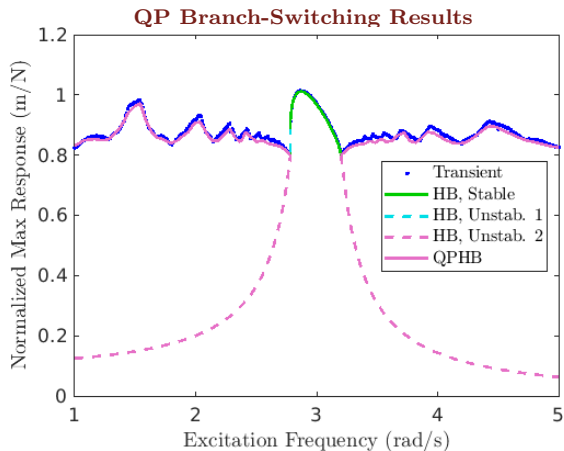
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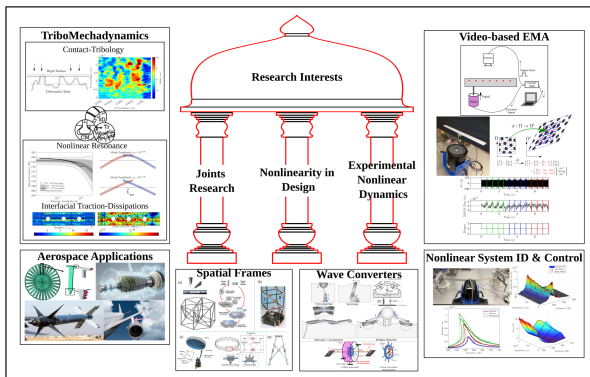
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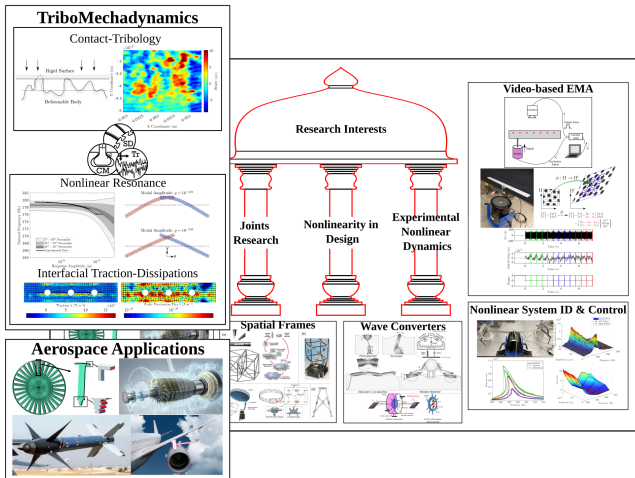
3. Nonlinear Dynamics of Jointed Structures

3.3. Research Plan Overview



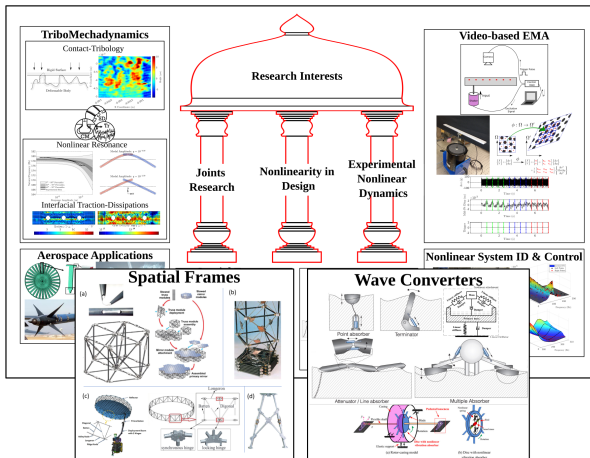
3. Nonlinear Dynamics of Jointed Structures

3.3. Research Plan Overview



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3. Nonlinear Dynamics of Jointed Structures

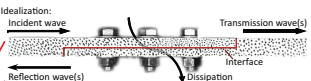
3.3. Research Plan Overview

Wave-Based Analysis of Aerospace Structures

NSF GOALI Proposal
(submitted, '22) with **MAC & NESC, USA**



Exploded view of Orion

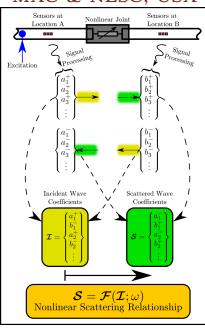


Idealization:
Incident wave, Reflection wave(s), Transmission wave(s), Interface, Dissipation

- Component 1: Joint Modeling Framework of Single Joints (Analytical & Data-Driven)
- Component 2: System Level Modeling with Many Joints
- Component 3: Experimental Validation

Goal:

Component Model
Wave-Based Frame Model
Component Model



Sensors at Location A Nonlinear Joint Sensors at Location B

Excitation → Signal Processing → Signal Processing

Incident Wave Coefficients: $\begin{bmatrix} a_1 \\ a_2 \\ a_3 \\ \vdots \end{bmatrix}$

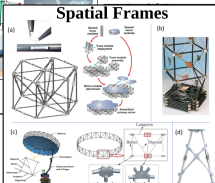
Scattered Wave Coefficients: $\begin{bmatrix} b_1 \\ b_2 \\ b_3 \\ \vdots \end{bmatrix}$

Transfer Matrix: $\mathcal{T} = \begin{bmatrix} t_{11} & t_{12} \\ t_{21} & t_{22} \end{bmatrix}$

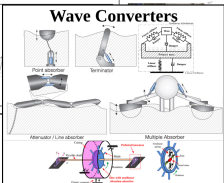
Nonlinear Scattering Relationship: $\mathbf{S} = \mathcal{F}(\mathcal{T}; \omega)$

Space Applications

Spatial Frames



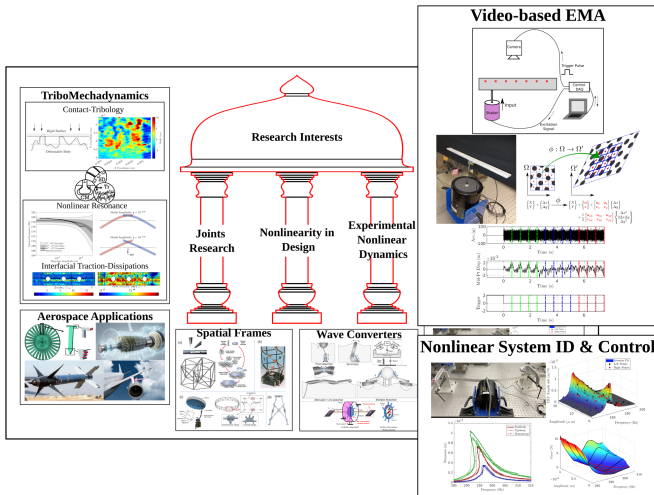
Wave Converters



Nonlinear System ID & Control

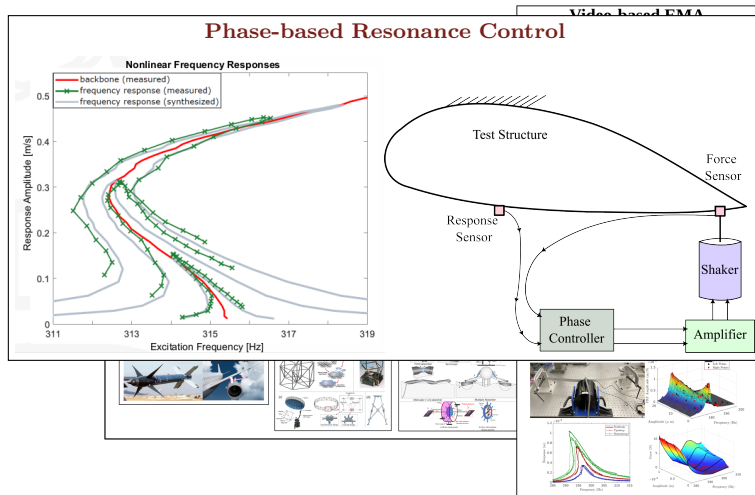
3. Nonlinear Dynamics of Jointed Structures

3.3. Research Plan Overview



3. Nonlinear Dynamics of Jointed Structures

3.3. Research Plan Overview



3. Nonlinear Dynamics of Jointed Structures

3.4. Focus on Free and Open Source Software

- Big proponent of Free and Open Source Computing!



Computational

Reduced-Order Modeling

[5] Balaji, *MSSP* (2021) 149, 107249

Wave-Based Jointed Structure Modeling

[8] Balaji, Brake, Leamy, *NLDyn* (2022a) 1-16
 [9] Balaji, Brake, Leamy, *NLDyn* (2022b) 1-27

Computational Nonlinear Dynamics

[12] Woiswode, Balaji, et al., *MSSP* (2020) 136, 106503

Joint Dynamics, Multi-Scale FE Contact Modeling

[1] Balaji, Chen, Brake, *MSSP* (2020) 136, 106503
 [2] Balaji, Brake *MSSP* (2019) 126, 42-64
 [3] Mathis, Balaji, et al. *AMR* (2020) 72 (4)
 [4] Balaji (2021) PhD Thesis, Rice University

Nonlinear Modal Analysis

[6] Balaji, Brake, *CompStruc* (2020) 230, 106184
 [7] Balaji, Krishna, Padmanabhan *JSV* (2018) 422, 526-541

System Identification

[10] Balaji, et al., *Vibration* 3 (3), 22 (2020)
 [11] Balaji, et al., (Under Prep.)

Experimental

Modal Testing

[14] Balaji, et al. *IMAC* (2020)

Long-Term Wear Evolution Studies

[15] Balaji, Smith, Brake *IMAC* (2023)

Video-Based Vibration Testing

[16] Balaji, Jermaine, Brake *ISMA* (2022)

[16] Balaji, Smith, Brake *IMAC* (Under Prep.) [17] Balaji, Jermaine, Brake *ISMA* (U.P.)

Ongoing...

Quasi-Periodic Nonlinear Dynamics

[18] Balaji, Groß, Krack (Under Prep.)
 [19] Balaji, Groß, Krack (Under Prep.)

Nonlinear Resonance Control

Stochastic Dynamics

<http://ae.iitm.ac.in/nidish>



Thank You!
 nidish@iitm.ac.in

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- N. N. Balaji and M. R. Brake. “A Quasi-Static Non-Linear Modal Analysis Procedure Extending Rayleigh Quotient Stationarity for Non-Conservative Dynamical Systems”. *Computers & Structures*, **230**, (Apr. 2020), pp. 106184. ISSN: 00457949. DOI: [10.1016/j.compstruc.2019.106184](https://doi.org/10.1016/j.compstruc.2019.106184). URL: <https://linkinghub.elsevier.com/retrieve/pii/S0045794919315160> (cit. on p. 25).
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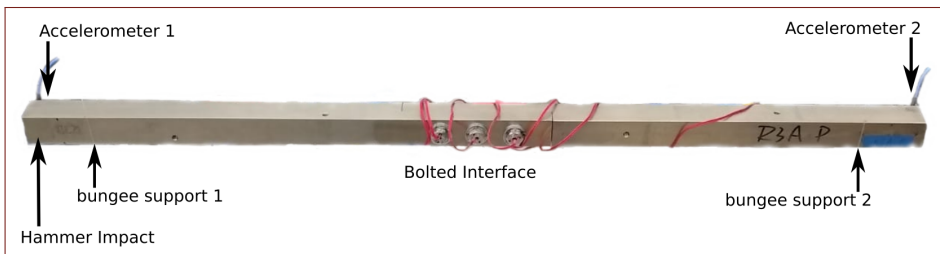
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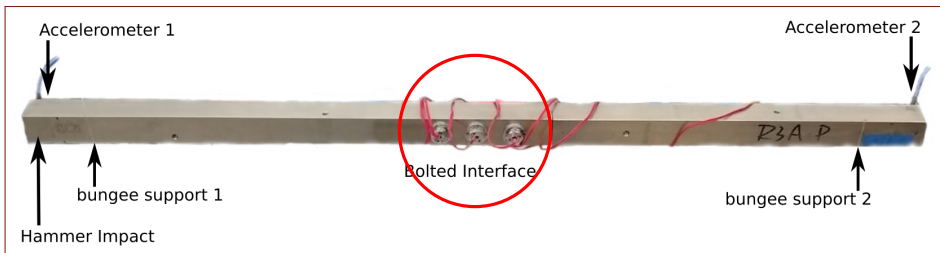
5. Experimental Details

Hammer Impact Testing Setup




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Hammer Impact Testing Setup




5. Experimental Details


Accelerometer



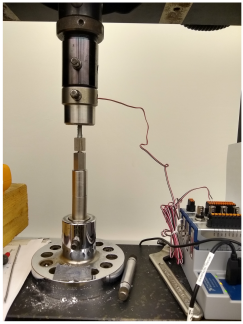
Tensile Test Calibration

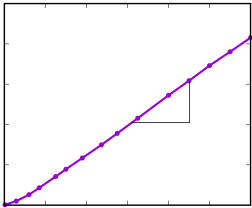
Accelerometer 2





Instrumented Bolts

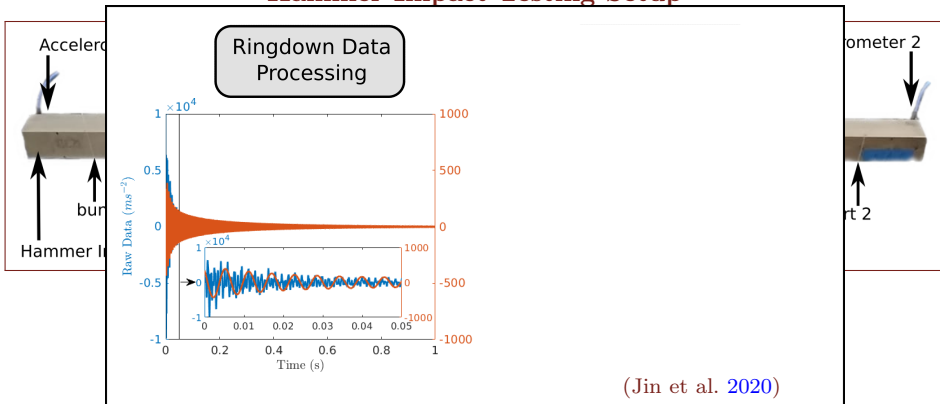




Force-Strain Characteristic

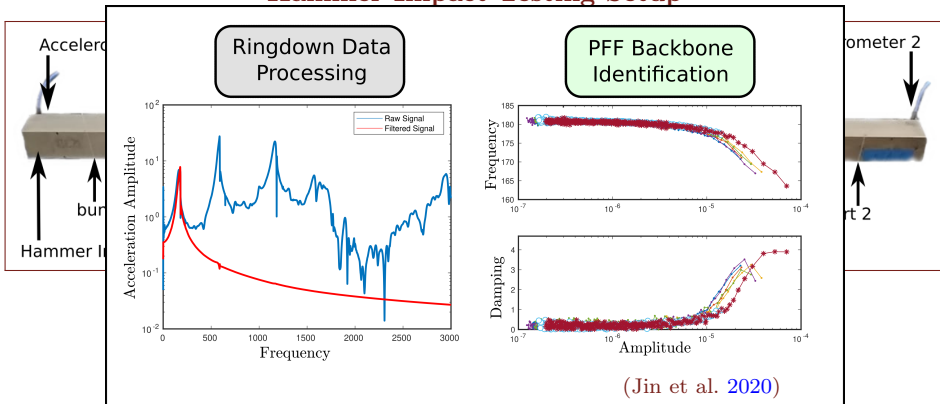
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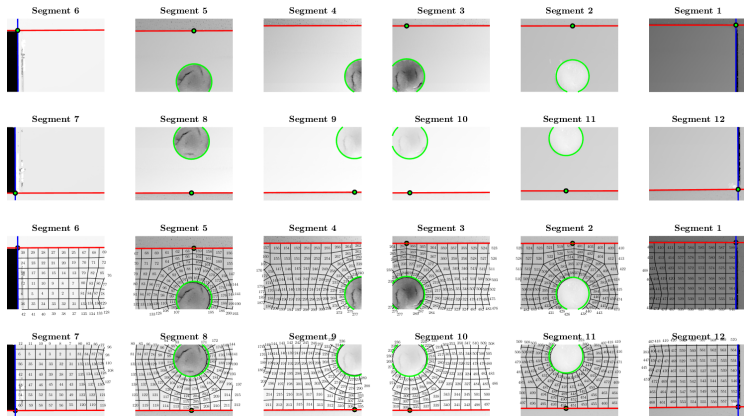
Interface Scanning and Processing



The Keyence VR-5100 White Light Interferometer

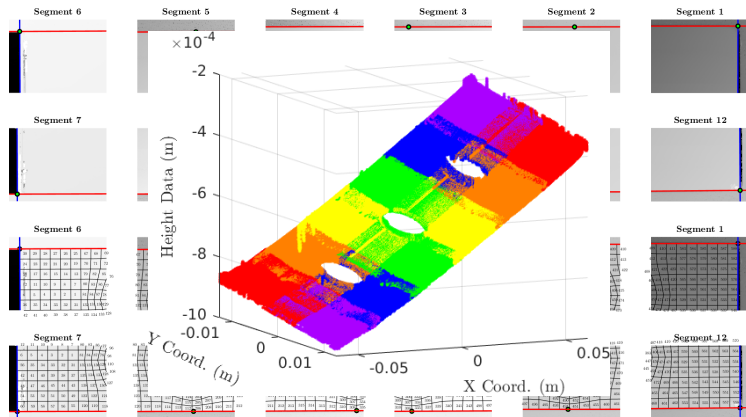
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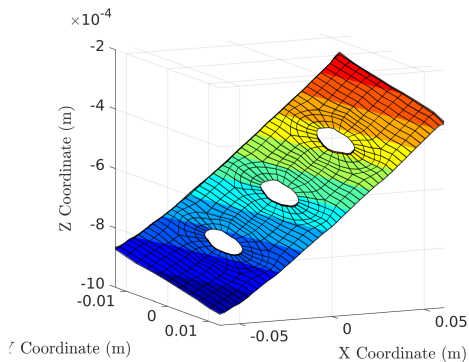
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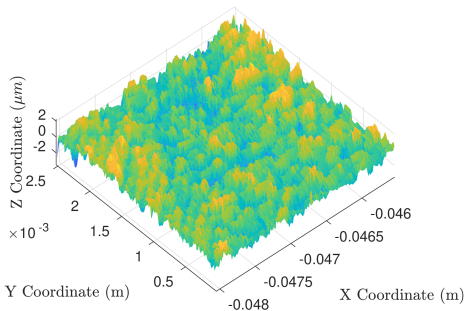


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Interface Scanning and Processing



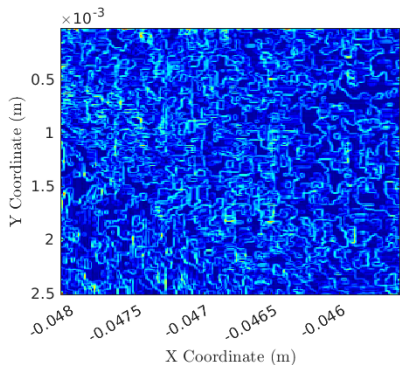
Meso-scale Topography



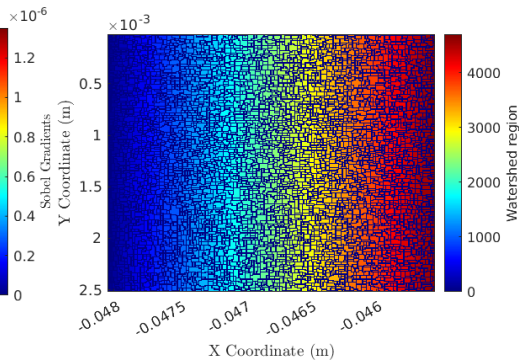
Micro-scale Asperity Distribution

5. Experimental Details

Interface Scanning and Processing



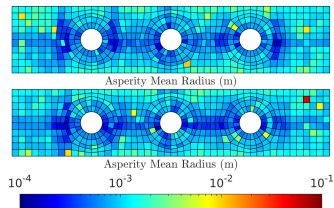
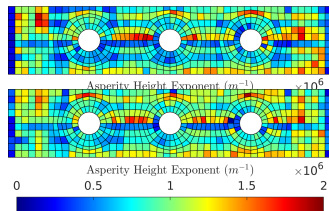
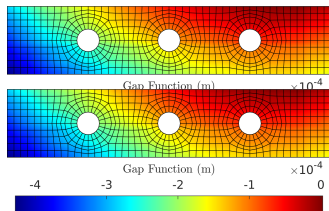
Sobel Gradients



Watershed Regions

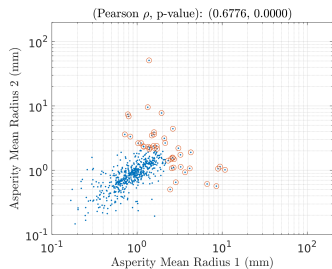
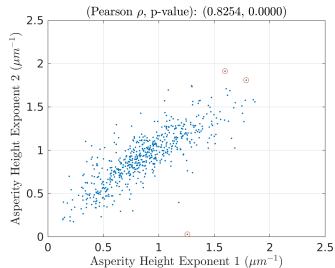
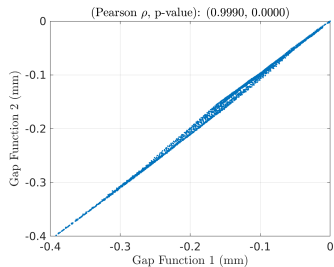
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Contact Parameter Estimation

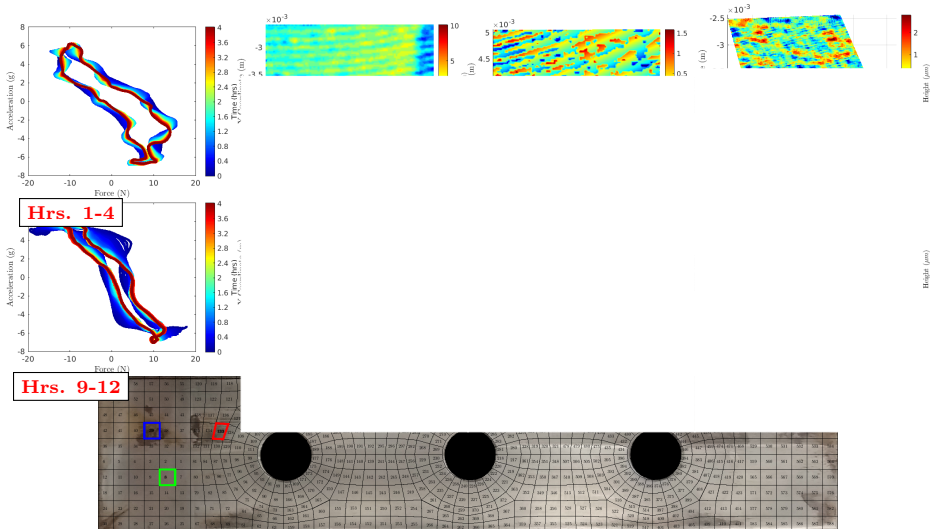


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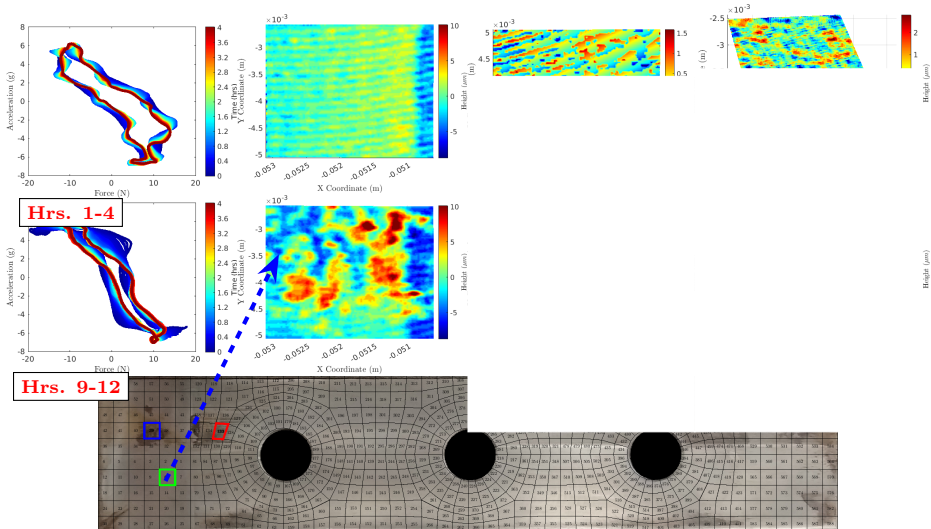
Contact Parameter Estimation



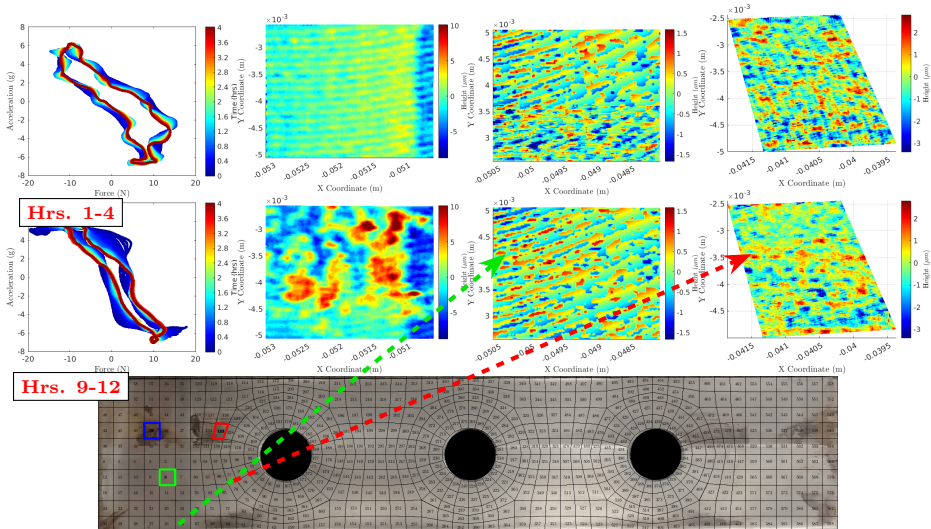
6. Wear Evolution



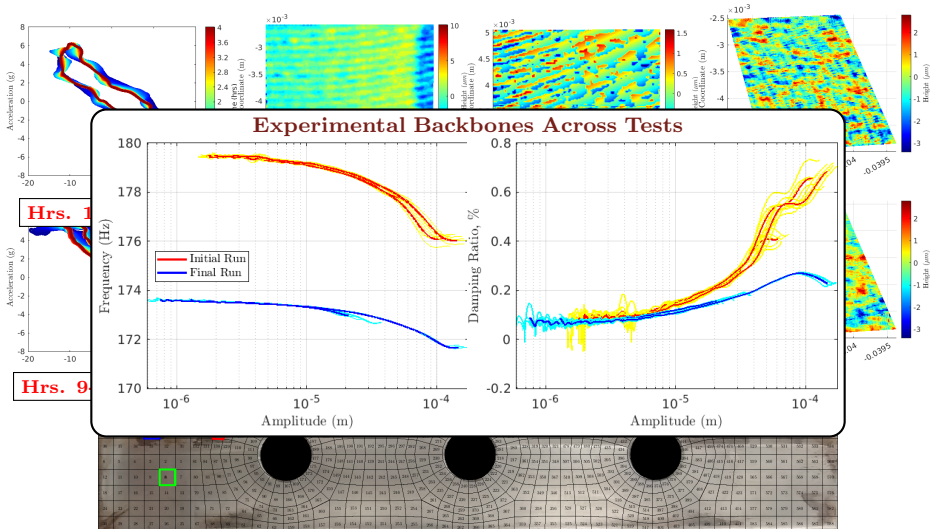
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7. Stochastic Modeling through Polynomial Chaos Expansion

- The Polynomial Chaos Expansion (PCE) approach is adopted for the stochastic modeling purpose (Wiener 1938; Sudret 2008)

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$$y \leftarrow f(x) \quad x \in \mathcal{D} \sim w(\cdot) \quad (\text{pdf: } w(x))$$

- The family of polynomials $\psi_n(x)$ that are orthogonal with respect to the inner product weighted by $w(x)$ are chosen as the bases for the PCE

$$\langle \psi_n, \psi_m \rangle = \int_{\mathcal{D}} \psi_n(x) \psi_m(x) w(x) dx = \mathbb{E}[\psi_n \psi_m] = \delta_{mn}.$$

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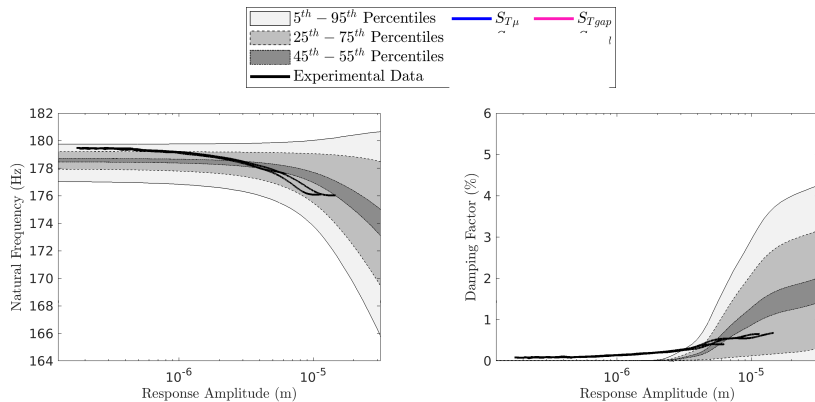
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- The Polynomial Chaos Expansion of y is written as

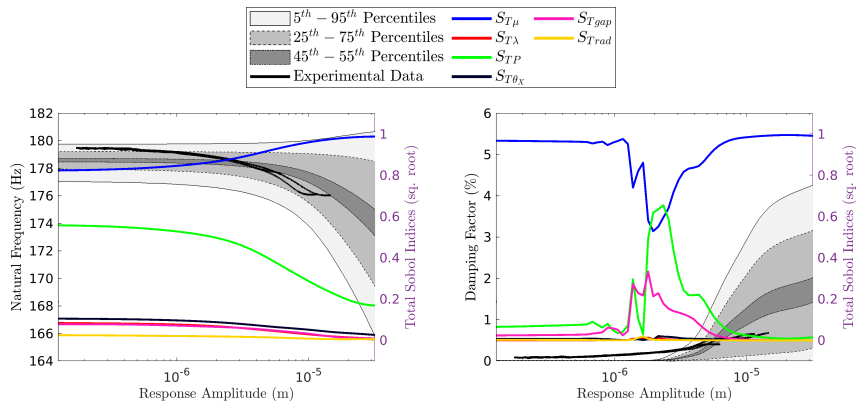
$$\hat{y} = f_0 + \sum_{i=1}^N f_i \psi_i(x) \rightarrow \hat{y} = f_0 + \sum_{i=1}^{N_1} \sum_{j=1}^{N_2} f_{ij} \psi_i^{(1)}(x_1) \psi_j^{(2)}(x_2).$$

Variance Decomposition

8. PCE Results



8. PCE Results



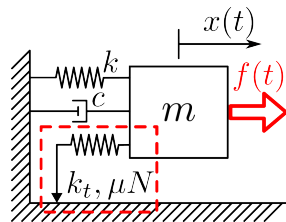
9. A Self-Excited SDOF Benchmark

- A representative problem is analyzed

$$m\ddot{x} - c\dot{x} + kx + f_{nl}(x, \dots) = \frac{F}{2}e^{j\Omega t} + c.c.$$

using a Harmonic Balance approach, with the Fourier ansatz:

$$x(t) = \sum_{k \in \mathcal{H}} U_k e^{jk\Omega t} + c.c.$$



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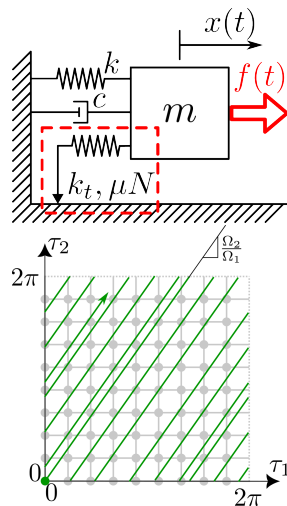
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$$x(t) = \sum_{\underline{k} \in \mathcal{H}} U_{\underline{k}} \exp(i(k_1 \tau_1 + k_2 \tau_2))$$



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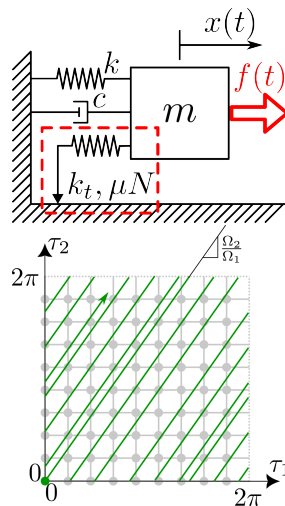
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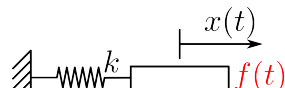
Evaluation of Frictional Forces

$$f_i = \begin{cases} \overbrace{k_t(x_i - x_{i-1}) + f_{i-1}}^{f^{(sp)}} & \text{stick} \\ \mu N \text{sign}(f^{(sp)}) & \text{slip} \end{cases}$$



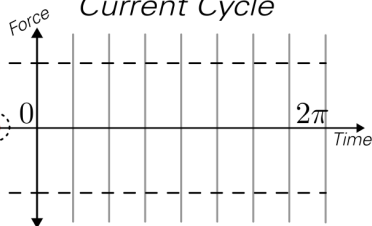
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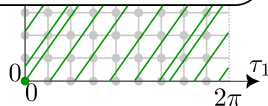
Previous Cycle

Current Cycle



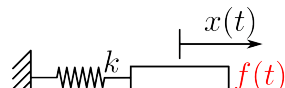
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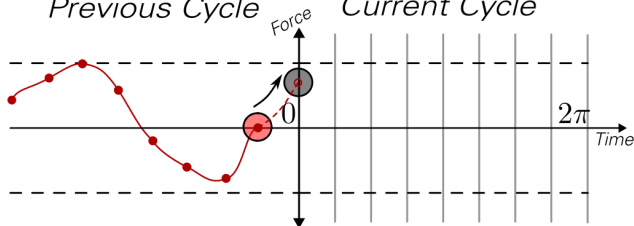
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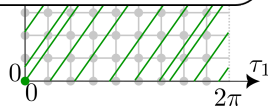
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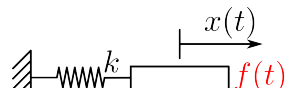
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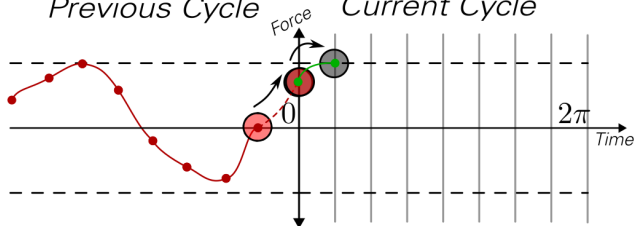
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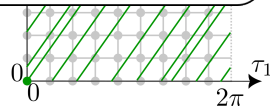
Previous Cycle

Current Cycle



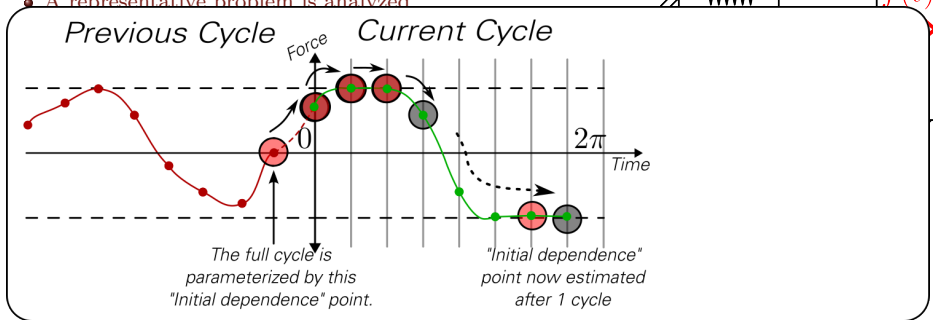
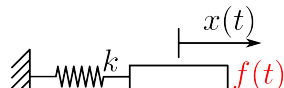
Evaluation of Frictional Forces

$$f_i = \begin{cases} \overbrace{k_t(x_i - x_{i-1}) + f_{i-1}}^{f^{(sp)}} & \text{stick} \\ \mu N \text{sign}(f^{(sp)}) & \text{slip} \end{cases}$$



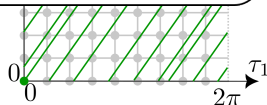
9. A Self-Excited SDOF Benchmark

- A representative problem is analyzed



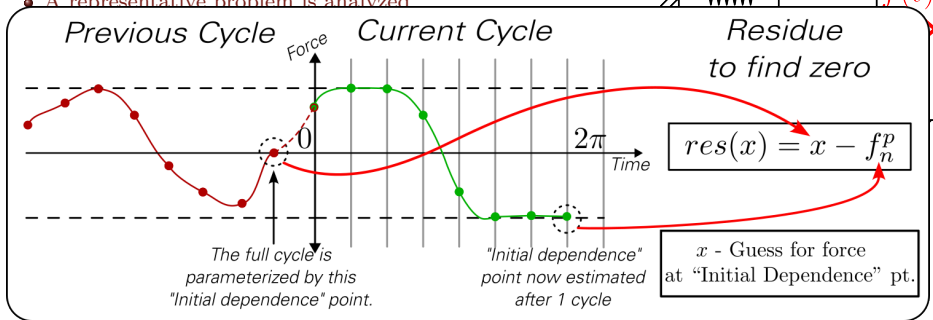
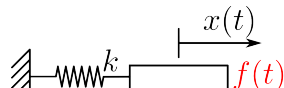
Evaluation of Frictional Forces

$$f_i = \begin{cases} \overbrace{k_t(x_i - x_{i-1}) + f_{i-1}}^{f^{(sp)}} & \text{stick} \\ \mu N \text{sign}(f^{(sp)}) & \text{slip} \end{cases}$$



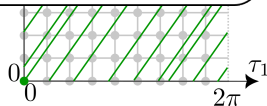
9. A Self-Excited SDOF Benchmark

- A representative problem is analyzed

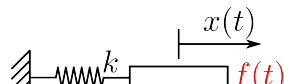


Evaluation of Frictional Forces

$$f_i = \begin{cases} \overbrace{k_t(x_i - x_{i-1}) + f_{i-1}}^{f^{(sp)}} & \text{stick} \\ \mu N \text{sign}(f^{(sp)}) & \text{slip} \end{cases}$$



9. A Self-Excited SDOF Benchmark



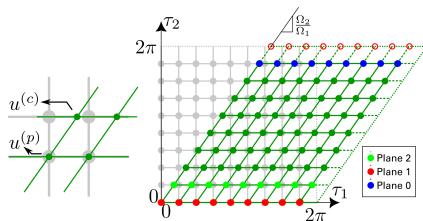
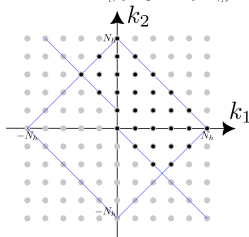
A representative problem is analyzed

QPHB Hyper-Time Marching (see Balaji, Groß, and Krack 2023, Under Preparation)

$$x(t) = \sum_{\underline{k} \in \mathcal{H}} U_{\underline{k}} \exp(i(k_1 \tau_1 + k_2 \tau_2))$$

Harmonic Truncation

$$\begin{aligned} \mathcal{H} = \{ & (k_1, k_2) \mid |k_1| + |k_2| \leq N_h \\ & \cap [k_1 + k_2 > 0] \\ & \cap [(k_1 + k_2 = 0 \cap k_1 > 0)] \} \end{aligned}$$



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